# Computer Modeling of Non-stationary Conduction of Heat in Two-layer Plate

HANA CHARVÁTOVÁ, VLADIMÍR VAŠEK, \*PAVEL MOKREJŠ, MILOSLAV FIALKA Tomas Bata University in Zlín, Faculty of Applied Informatics, \*Faculty of Technology nám. T. G. Masaryka 5555, 760 01 Zlín CZECH REPUBLIC charvatova@fai.utb.cz http://www.utb.cz

*Abstract:* - In this paper we deal with study of a heat transport process in solids. Especially we focused on a problem of non-stationary conduction of heat in a two-layer plane plate. For this purpose we formulated mathematical model describing heating or cooling of a semi-infinite region. The analytical solution of this model we used for computer modelling of the mentioned process by use of mathematical software Maple.

In the second part of the paper we demonstrate modelling of computing of heating or cooling of the two-layer plane plate by use of the software application that we programmed for automatic computing of temperature fields in the solids during heating or cooling of the two-layer plane plate. Finally, we verified validity of the formulated problem by comparison of the computed data with computer simulation of the process by use of commercial software Comsol Multiphysics.

Key-Words: - Non-stationary heat conduction, Temperature field, Two-layer plate, Mathematical model

### **1** Introduction

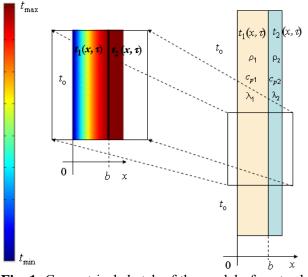
Many technological operations of the synthetic materials treatment are based on procedures of heat effect on the processed body. These operations are generally energy demanding. Therefore we deal with their optimization[1],[2]. For finding of an appropriate optimization method, it is necessary to get information about given process course. But these data are often hardly experimentally determined and in addition time-consuming. Therefore mathematical modeling and computer simulation of the studied process can be only alternative how this information to obtain.

In many cases, the mathematical model of given process is very specific and in addition depending on concrete conditions of the process. In this paper we will focus on a case of unsteady conduction of heat in the solid material. We will study problem of non-stationary conduction of heat in the two layers plate. In the following text we will formulate mathematical model of the mentioned process. Next we will present software application that we programmed for computing of unsteady temperature fields in the two-layer plate during its one-sided heating or cooling.

Finally we will show results that we obtained by comparison of the computed data with computer simulation of the process by use of the commercial software Comsol Multiphysics.

## **2 Problem Formulation**

In this section we will formulate mathematic model in the case of one-sided heating or cooling of twolayers semi-infinite wall in the region  $0 < b < \infty$ . Geometrical sketch of the problem you can see in Fig. 1.



**Fig. 1** Geometrical sketch of the model of unsteady heat conduction in two-layer plane plate

The plate with the initial constant temperature  $t_p$  will be exposed to the sudden one-sided heat action. Mathematical model of the process can be described by Fourier equation of the heat conduction (1) with initial and boundary conditions (2) – (7) [3], [4]:

$$\frac{\partial t_1}{\partial \tau}(x,\tau) = a_1 \frac{\partial^2 t_1}{\partial x^2}(x,\tau), \ \tau > 0, \ 0 < x < b$$
(1)

$$\frac{\partial t_2}{\partial \tau}(x,\tau) = a_2 \frac{\partial^2 t_2}{\partial x^2}(x,\tau), \ \tau > 0, \ b < x < \infty$$
(2)

$$t_1(x,0) = t_2(x,0) = t_p$$
 (3)

$$t_1(0,\tau) = t_o \tag{4}$$

$$\frac{\partial t_2(\infty,\tau)}{\partial x} = 0 \tag{5}$$

$$t_1(b,\tau) = t_2(b,\tau) \tag{6}$$

$$\lambda_{1} \frac{\partial t_{1}(b,\tau)}{\partial \tau} = \lambda_{2} \frac{\partial t_{2}(b,\tau)}{\partial \tau}$$
(7)

where  $a_1 = \frac{\lambda_1}{\rho_1 \cdot c_{p1}}$  or  $a_2 = \frac{\lambda_2}{\rho_2 \cdot c_{p2}}$  are thermal

conductivity of the first or second layer of the heated (cooled) solid.

The condition (3) is assumption of the initial constant temperature in the plate. The boundary condition (4) is assumption of the time independent temperature in the boundary of the first layer and surrounding fluid. The boundary condition (5) is assumption of semi-infinite region. The boundary conditions (6) and (7) are assumptions of perfect contact of the layers. The relations (8) and (9) describe analytical solutions of the model [3]. The equation (8) describes temperature fields in the first layer  $t_1(x, \tau)$ :

$$\frac{t_1 - t_p}{t_o - t_p} = \left[\sum_{n=0}^{\infty} h^n \left\{ \operatorname{erfc} \frac{(2n+1)b + x}{2\sqrt{a_1 \tau}} - h \cdot \operatorname{erfc} \frac{(2n+1)b - x}{2\sqrt{a_1 \tau}} \right\} \right]$$
(8)

Distribution of temperature in the second layer  $t_2(x, \tau)$  can be described by equation (9):

$$\frac{t_2 - t_p}{t_o - t_p} = \frac{2K_{\varepsilon}}{1 + K_{\varepsilon}} \sum_{n=1}^{\infty} h^{n-1} erfc \left( \frac{x - b + (2n-1)K_a^{-\frac{1}{2}}b}{2\sqrt{a_2\tau}} \right)$$
(9)

where

$$h = \frac{1 - K_{\varepsilon}}{1 + K_{\varepsilon}} \tag{10}$$

$$K_{a}^{-\frac{1}{2}} = \sqrt{\frac{a_{2}}{a_{1}}} = \sqrt{\frac{\lambda_{2}\rho_{1}c_{p1}}{\lambda_{1}\rho_{2}c_{p2}}}$$
(11)

$$K_{\varepsilon} = \frac{\lambda_1}{\lambda_2} \sqrt{\frac{\lambda_2 \rho_1 c_{p1}}{\lambda_1 \rho_2 c_{p2}}}$$
(12)

#### 2.1 List of symbols

- *a* thermal diffusivity,  $m^2.s^{-1}$ ;
- $c_p$  specific thermal capacity, J.kg<sup>-1</sup>.K<sup>-1</sup>;
- *t* temperature of the heated (cooled) body, °C;
- $t_o$  ambient temperature, °C;
- $t_p$  initial temperature of the heated (cooled) body, °C;
- *x* space coordinate, m;
- *b* thickness, m;
- $\lambda$  thermal conductivity, W.m<sup>-1</sup>.K<sup>-1</sup>;
- $\rho$  density, kg.m<sup>-3</sup>;
- $\tau$  time, s;
- 1 properties of the first layer;
- 2 properties of the second layer.

#### **3** Computer modeling the process

We computed temperature fields (8) and (9) in the mathematical software Maple user environment. In the Fig. 2 we show course the temperature field in the plate during heating. In the Fig. 3 you can see course the temperature field in the plate during cooling.

For simplification of computation of the temperature fields we also programmed interactive application for modelling of the above described process of the heat action course in the mathematic software Maple environment. We made the application in the Maplet form which enables us to insert required input parameters, automatic compute and display temperature fields as both 3D graphics  $t(x, \tau)$  and 2D graphics t(x) in the required time of the process. Our software application can also compare the temperature fields in various time of the process and export the displayed graphics. In the Fig. 4 we present user interface of the application.

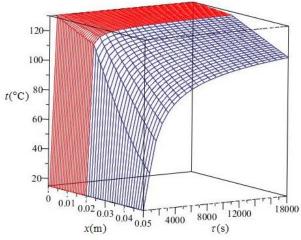
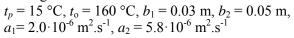
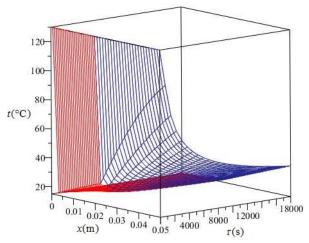


Fig. 2 Temperature field in the two-layer plate during heating





**Fig. 3** Temperature field in the two-layer plate during cooling

 $t_p = 160 \text{ °C}, t_0 = 15 \text{ °C}, b_1 = 0.03 \text{ m}, b_2 = 0.05 \text{ m}, a_1 = 2.0 \cdot 10^{-6} \text{ m}^2 \cdot \text{s}^{-1}, a_2 = 5.8 \cdot 10^{-6} \text{ m}^2 \cdot \text{s}^{-1}$ 

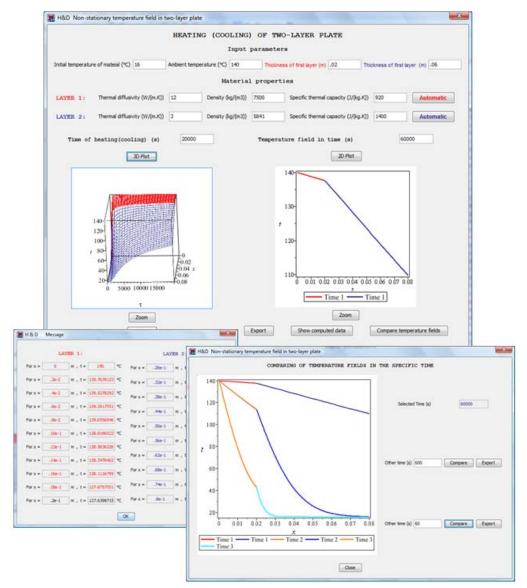


Fig. 4 Show of user environment of the programmed software application

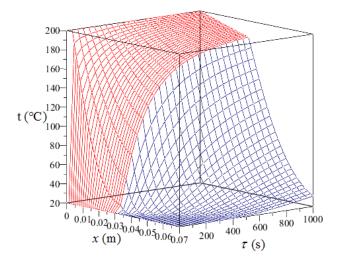
# **3.1** Comparison of the computed data with computer simulation of the process

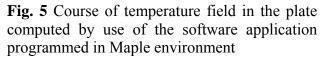
For verification of the formulated model validity we compared temperature fields computed by Maple with simulation by use of the commercial software Comsol Multiphysics.

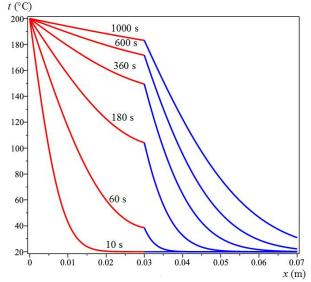
In the following Fig. 5 – 8 we show simulated and computed data that we obtained by testing of heating of two-layer plate under the same conditions. Initial temperature of the plate is 20 °C, ambient temperature is 200 °C. Thickness of the first layer is 0.03 m, thickness of the second layer is 0.04 m. Thermal conductivity of the first layer is  $2.15 \cdot 10^{-6}$  m<sup>2</sup>.s<sup>-1</sup>. Thermal conductivity of the second layer is  $3.06 \cdot 10^{-6}$  m<sup>2</sup>.s<sup>-1</sup>.

In the Fig. 5 and Fig. 6 you can see temperature fields that we have computed by use of our software application programmed in Maple environment. In the Fig. 5 we show course of the temperature field for 1000 seconds of the heating.

The Fig. 6 represents temperature fields in time 10 s, 60 s, 180 s, 360 s, 600 s and 1000 s.



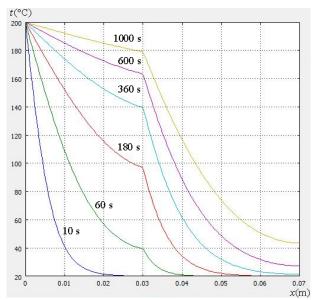




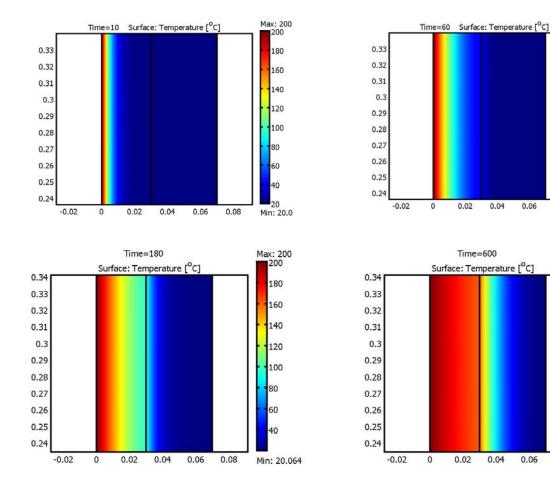
**Fig. 6** Temperature fields in the plate computed by use of the software application programmed in Maple environment in time 10 s, 60 s, 180 s, 360 s, 600 s and 1000 s.

In the Fig. 7 and Fig. 8 we show temperature fields in time 10 s, 60 s, 180 s, 360 s, 600 s and 1000 s simulated in the Comsol Multiphysics.

It is evident, that in the both cases, the temperature fields have a similar course, which confirms possibility to use our software application for modelling of the mentioned process.



**Fig. 7** Temperature fields in the plate simulated by Comsol Multiphysics in time 10 s, 60 s, 180 s, 360 s, 600 s and 1000 s



**Fig. 8** Distribution of temperature in section of the plate simulated by Comsol Multiphysics in time 10 s, 60 s, 180 s, and 600 s

#### 4 Conclusion

We formulated mathematical model that is suitable for description of heating or cooling of two-layer plane plate in the case of the semi-infinite region.

Analytical solution of the model we used for programming of the software application that can compute temperature fields and enabled to get quick notion about studied process course. The software application can compute data necessary for optimization of many technological operations which are based on the similar mechanism as we described in this paper.

Validity of the formulated model we verified by comparison of the computed data with computer simulation of the process by use of Comsol Multiphysics. The obtained results proved possibility to use our software application for modeling of the real technological processes.

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0.08

Max: 200

180

160

140

120

100

80

60

40

Min: 20.0

Max: 200

200

180

160

140

120

100

80

60

40

Min: 27.358

0.08

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